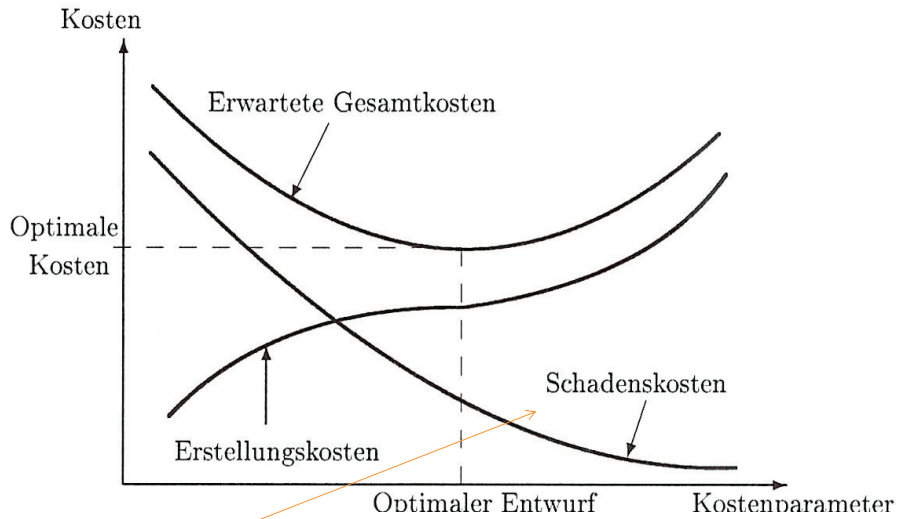
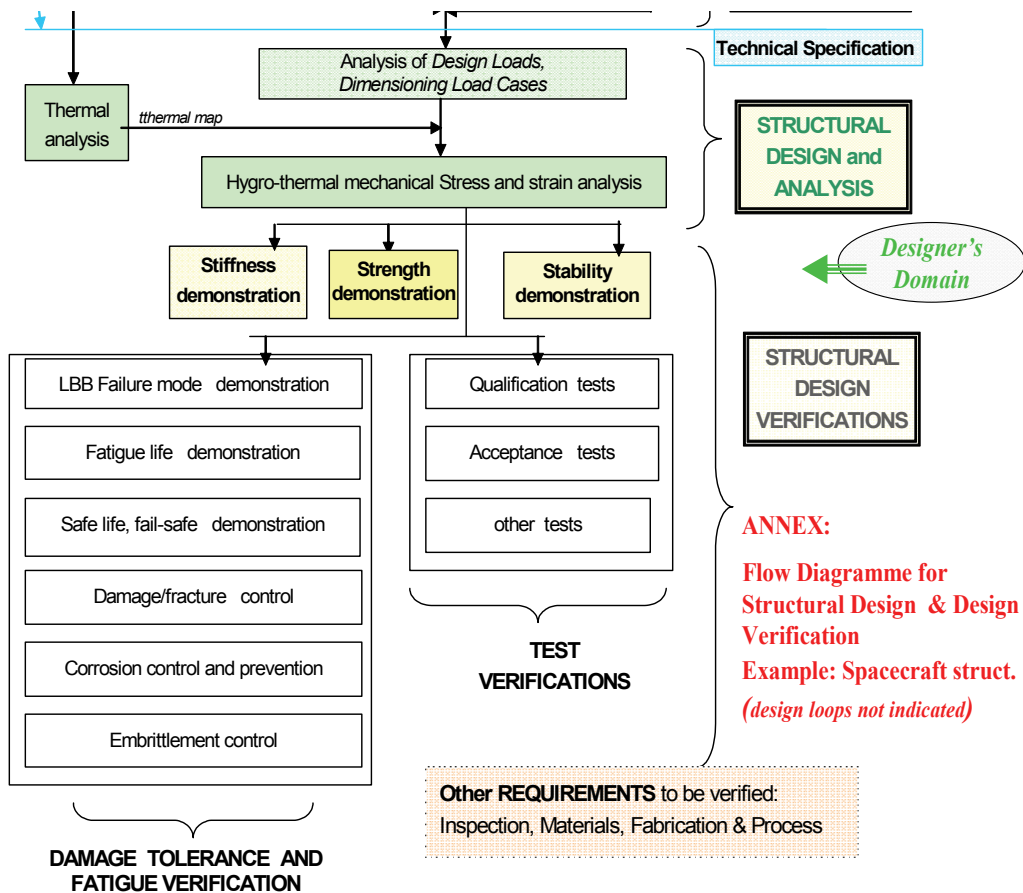


ANNEXES:

Cost → Minimum  
Quality → Maximum



Risk = amount of cost incurred in the case of later failure x probability of its occurrence



# Fundamentals in the Development of *Reliable Structural Components?*

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## Abstract

Modern light-weight structures are the result of an optimisation compromise between all the product's functional requirements (e.g. stiffness, strength) and all the operational requirements (e.g. lifetime).

Very often the strength requirement determines the mass. Design driving hereby are the material properties and the failure conditions for fracture and yielding which normally meets a functional design verification).

Responsible for the quality of a structural component achieved under the minimum mass requirement are a qualified analysis procedure, a reliable data input including the dimensioning load cases, and the safety concept. At present, usually a deterministic safety concept is applied which employs factor of safety values  $f_{yield}$ ,  $f_{ultimate}$ .

Special task of the designer is the development of a so-called robust structure which does not essentially change its behaviour under the usual scatter of the stochastic parameters associated with shortcomings for the fulfilment of its functions.

Due to above aspects the following elements will be dealt with

1. *Problem Description*
2. *Introduction to Design Dimensioning and Design Verification (Nachweis)*
3. *Safety Concept Applied*
4. *Input of Appropriate Properties for Linear and Non-linear Analysis*
5. *Design Limit Loads, Dimensioning Load Cases, Load Interaction Failure*
6. *Link Deterministic to Probabilistic Design.*

Also, the definitions of some notions such as Design Limit Load (Sichere Last) and Dimensioning Load Case are presented.

Finally, an application of the reliability analysis to the ARIANE 5 Boosters shall demonstrate the usefulness of a selective probabilistic design.

**Keywords:** Robust Design, Optimization, Stochastic

## General

Robustness and reliability are cross-linked to some extent. Non-robustness with respect to the requirements in the Technical Specification may cause heavy changes of the reliability level. The *risk*, defined as *costs in case of failure times the probability the failure may occur* is higher than in case of a robust design optimisation.

Stochastic design parameters are termed uncertain basic variables, the size of which is uncertain (before realization of them) and random (after realization). Uncertainty and randomness can be described by a distribution law with its distribution parameters, e.g. in case of a *parent* normal distribution (Gauß) the statistical parameters mean  $\mu$  (of *test sample*,  $\bar{x}$ ) and standard deviation  $\sigma$  (of test sample,  $s$ ).

The usual deterministic optimisation procedure for a structural component is an optimisation in respect of the different actual failure modes. A distinct set of design parameters is optimized in the design space with respect to an optimal state such as for the failure modes buckling, fracture, limited strains or a natural frequency.

All the possible (failure) limit states are not met by the deterministic set of optimal design parameters by a certain distance due to the required factors of safety (FoS) which are usually used as Design Limit Load-increasing factors. Unfortunately, this distance is not quantifiable.

However, the probabilistic optimisation provides the designer with a measure for the distance by giving him a number for a *reliability = 1 - failure probability*. Of course, this number is a fictitious one because it depends on the quality of the used model and the input.

A deterministic-optimal set of design parameters does not consider the sometimes heavily different coefficients of variation  $CoV = \sigma / \mu$  of the driving design parameters and the probability of their combined appearance (joint probability of failure). This means the values, the stochastic design parameters may take in the design space. Therefore normally, the set of deterministically derived optimal design parameters will be different to that of a probabilistically derived one with for instance the consequence that the nominal values in the drawing will be different.

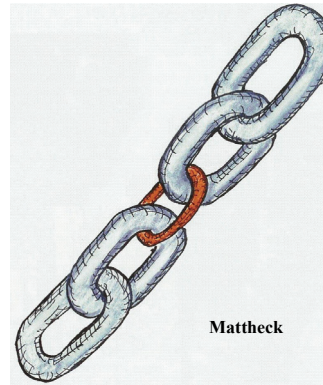
In probabilistics-based optimisation no FoS are utilized but the distributions of the stochastic design parameters are applied (for loads usually an extreme value distribution is assumed). Essential aspect of this type of optimisation is the sensitivity this means the influence of a design parameter on the objective function such as a collapse load or a mass value. The lower the change of the sensitivity measures is -in case of a change of the scatter of a design parameter- the more robust the design is. The knowledge of such sensitivity measures helps with management decisions, e.g. Which of the geometrical tolerances have less influence and can therefore be met simpler and fulfilled cheaper, however, with keeping the same reliability?

## Fundamentals in the Development of *Reliable Structural Components*

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### Motivation:

Industry looks for *robust & reliable*  
*analysis procedures*  
in order to replace  
the expensive 'Make and Test Method'  
as far as reasonable.



### TASK:

- ▶ Sort out weakest link in design process  
which involves highest uncertainty.  
→ Just then: a Qualified Prediction  
Method is achievable.

## Fundamentals in the Development of *Reliable Structural Components*

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**Contents of Presentation:** (35 min + 5 discussion time)

1. Problem Description
  2. Introduction to *Design + Analysis* and *Design Verification*
  3. Safety Concept Applied
  4. Input of Appropriate Properties for Linear and Non-linear Analysis
  5. Design Limit Loads, Dimensioning Load Cases, Load Interaction Failure
  6. Link Deterministic to Probabilistic Design
- Conclusions + some Comments

# 1 Problem Description

## 1.1 Definitions: Robust Design and Structural Reliability

### **Robust Design:**

Entwurf einer Struktur,  
 der bei den normalen Streuungen der Entwurfsparameter  
 keine so große Veränderung des Tragverhaltens nach sich zieht,  
 so dass lediglich nur eine tolerierbare Gefahr  
 für die Nicht-Erfüllung der Funktionsanforderungen vorhanden ist .

### **Struktur-Zuverlässigkeit:**

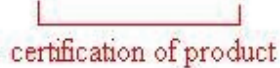
Zuverlässigkeit eines Struktur-Bauteils ist die Fähigkeit  
 während einer vorgegebenen Zeit  
 (mit einer bestimmten Wahrscheinlichkeit P)  
 vorgegebene Funktionsanforderungen zu erfüllen.

# 1 Problem Description

## 1.2 Development Phases and Associated Topics

Tight schedule constraints lead to a so-called 'Success oriented Development' Logic with its (theoretically) 'One Phase Design Development' idea

Phase	DESIGN	Design Analysis	Test
1	concept	conceptual	sizing
2	design development	preliminary	design development tests
3		critical (final)	
4	qualification	accepted	experimental design verification
5	production		

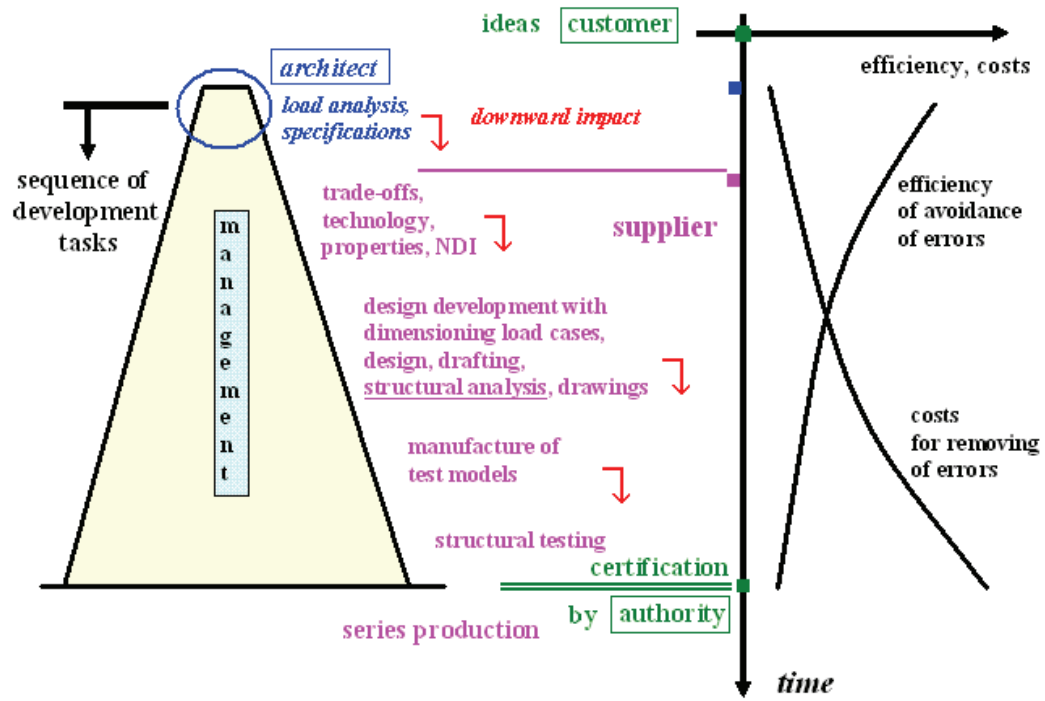

  
 certification of product

**Topics** which are mandatory to be considered when aiming at a 'SoD' Logic are :

- excellent *Technical Specifications* of the to be developed product
- a *consistent design philosophy* incl. safety concept, margins, nonlinear analyses, ...
- *Simultaneous Engineering*
- a *practical risk judging*
- *accurate modelling* incl. design allowables , mean ( $\sigma, \epsilon$ )-curves, failure hypothesis, mean geometry, ...
- a *back-up design solution* or realization of Design Target "Robustness"...

## 1 Problem Description

### 1.3. The Cost Pyramide in Development of a Structural Design



## 2 Introduction to Design Dimensioning and Design Verification

### 2.1 Design Requirements

Design must fulfill all *design requirements*:

- mass, production cost and life cycle cost, geometry
- loads, temperature, moisture, chemical environment
- limits of deformation, lifetime, leakage, eigenfrequency,
- **strength, stiffness, dimensional stability, buckling...**

topic

## 2 Introduction to Design Dimensioning and Design Verification

### 2.2 Closed Design analysis and Design Verification procedure

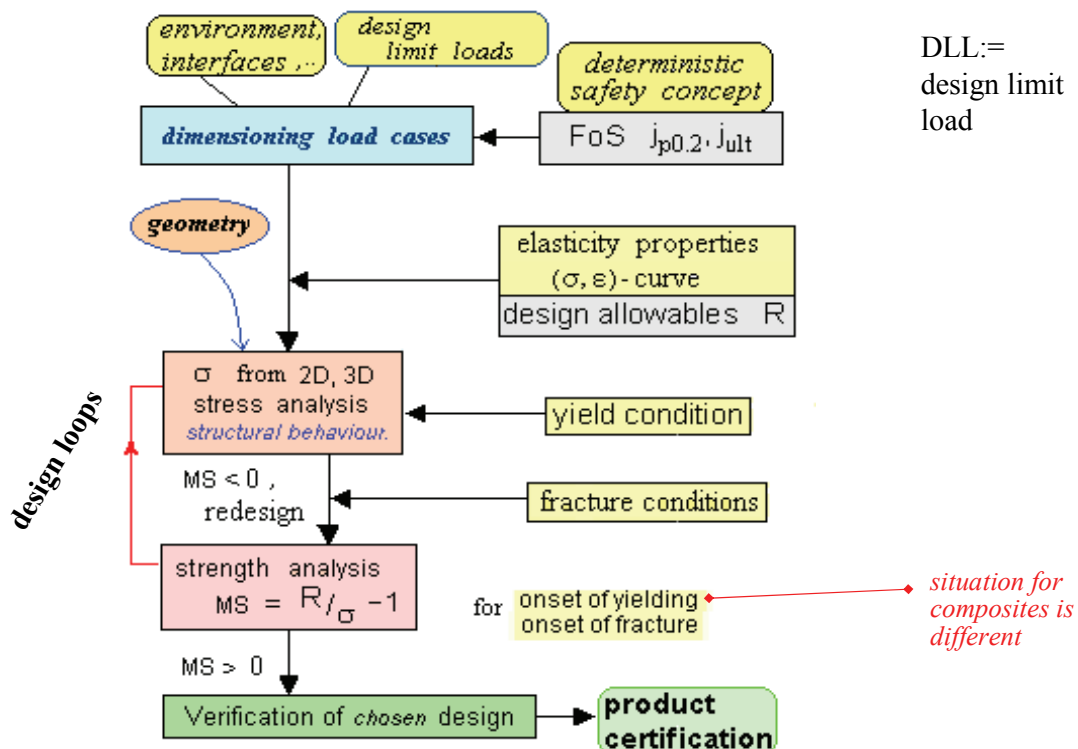
#### EXAMPLE: Shell buckling:

1. Determination of buckling load and buckling modes for the *idealized perfect* shell
2. Investigation of realistic imperfections for area pressure and local single forces
3. Fixation of most unfavourable imperfections
4. Prediction of buckling load and buckling modes for *imperfect* shell
5. Measurement of imperfect shell geometry
6. Simulation and model improvement
7. *Improved design concept (buckling load determination).*

► Consistent procedures help to minimize uncertainty

## 2 Introduction to Design and Design Verification

### 2.3 Static Structural Analysis Flow Chart (isotropic case)



## 2 Introduction to Design Dimensioning and Design Verification

### 2.4 Demonstration of Static Strength Capability

---

The Design's strength is demonstrated if

- no relevant strength failure (= limit state of a failure mode) is met and
- all dimensioning load cases are respected, by a

positive Margin of Safety  $MS > 0$

or a Reserve Factor  $f_{Res} = MS + 1$  for the prevailing failure modes.

**Assumption in usual deterministic procedure:**

**Worst case scenario**

with respect to loading, temperature and moisture,  
and undetected damage.

## 2 Introduction to Design and Design Verification

### 2.5 Demonstration of Static Strength Capability

---

Strength Demonstrations are a subset of all *Design Verifications* ('Nachweise')

- 1) onset of detrimental deformation (global yielding, yield failure)  
 equivalent stress at *DYL* smaller yield strength  $\sigma_{eq} \leq R_{p0.2}$
  - 2) onset of fracture (*final failure*)  
 equivalent stress at *DUL* smaller fracture strength  $\sigma_{eq} \leq R_m$
- functional limit →
- fracture limit →
- Normal Fracture  
or  
Shear Fracture

$R_{p0.2}$  = a practical engineering fixation, stands for a remaining plastic strain of  
 $\epsilon_{pl} = 0.2\%$  or a maximum detrimental deformation!  
 $\sigma_{eq}$  = equivalent stress



### 3 Safety Concept Applied

#### 3.1 The Uncertainties in the Design incl. Design Load Derivation

---

Uncertainties can be found

- *in the area of data input*
- *in the analysis of the whole modelling process*
- *in manufacture.*

Stages with uncertainties comprise

- \* *load analysis, testing and test data evaluation,*
- \* *choice of non-linear stress-strain curve and safety concept,*
- \* *choice of yield condition and fracture conditions,*
- \* *structural analysis procedure, and finally the*
- \* *determination of the MS value itself.*

*(loads, strength properties, geometry, elasticity properties, tolerances, imperfections, etc.)*

*NOTE: Here, uncertainties include inaccuracies as well as any simplifications in the design.*

### 3 Safety Concept Applied

#### 3.2 Types of Uncertainties

---

The nature of Uncertainty of *scattering design parameters* might be of

- *mechanical type* but also of
- *statistical type* (e.g. the way measurements are performed,
  - \* *lack of accurate information due to*  
*insufficient sample size in measurements of a specific design parameter,*
  - \* *limited observations or tests used for estimating the statistical distributions,*
  - \* *some uncertainty in the calculation model (e.g. solution procedure, mesh, ..)*  
*as well as in the*
  - \* *results provided by testing, evaluation of 'raw test data'.*

### 3 Safety Concept Applied

#### 3.3 The Uncertainties in the Design incl. Design Load Derivation

---

All these uncertainty sources contribute to the *overall Structural Risk* defined here arbitrarily as  
*Amount of costs (incurred in the case of later failure)*  
*x the probability that the distinct failure occurs in the structural part.*

### 3 Safety Concept Applied

#### 3.4 General Concepts

---

##### Safety Concept:

*concept, that implements structural reliability in design (safety is actually a wrong term but used).*

*Two formats are available for considering design uncertainties:*

- *The deterministic format accounts for design uncertainties in a lumped manner by enlarging the design limit loads by multiplication with FOS.*
- *The probabilistic format maps each single design parameter's uncertainty into a probability density function. Thereby, the joint probability of failure caused by a combination of design parameters can be considered.*

*NOTE: The joint probability of failure respects the combination of all scatter-caused varying design parameters*

*= Simplest form of the so-called Partial Safety Factor concept, being the simplest probabilistic safety factor concept !*

**Which is the actual safety concept in aerospace ?**

### 3 Safety Concept Applied

#### 3.5 Concept applied in Aerospace

---

**Actual safety concept in aerospace** = an *improved deterministic format*

- enlarges the deterministic loads (or stresses, if linear analysis is permitted) and
- causes a distance to the load resistance (or strengths).

*This distance represents the required positive margin of safety (MS).*

- discriminates *load uncertainties considering factors ( $K_M, K_P$ )* from *design uncertainties considering factors (FoS) !*

### 3 Safety Concept Applied

#### 3.6 General on (Design) Factors of Safety FoS

---

##### **Purpose of the Design FoS:**

Guaranty quality of the design and of the test  
in order to achieve  
a certain level of Structural Reliability for the (flight) hardware!

##### **Mind:**

- FoS are used to decrease the chance of failure by covering the uncertainties (affecting the risk of structural failure) of all the given variables outside the control of the designer which are primarily uncertainties in the statistical distribution of loads, uncertainties in manufacturing process, material strength properties .
- Missing accuracy in modelling, computing, or test data determination cannot be covered by the FoS !
- Assumption: Spacecraft standard is reached by contractor by validated design methods, qualified manufacturing processes etc.!
- Values for the FoS are different for cases such as :  
Manned, un-manned spacecraft and 'Design verification by Analysis only'.

*NOTE: The design risk is counteracted by the FoS.*

*Different industry, however, has different risk acceptance attitudes and apply differently high FoS values !*

### 3 Safety Concept Applied

Mit Arthurs HSB JAR Blatt checken

#### 3.7 Additional Factors in Design

Additionally utilized in design, is taken when the **sizing approach** is **complex**.

Such a factor accounts for specific uncertainties **linked to analysis difficulties**.

Such factors are *Fitting Factor, Welding Factor, Casting Factor,...* etc .

- FoS values are based on long term experience with
- structural testing (Composite experience is shorter)

### 3 Safety Concept Applied

#### 3.8 Example for a Factors of Safety (FoS) Table Draft

Structure type / sizing case	FOSY $j_{p0.2}$	FOSU $j_{ult}$	FOSY for verification 'by analysis only'	FOSU for verification 'by analysis only'	Design Factor	FOSY $j_{p0.2}$	FOSU $j_{ult}$	$j_{proof}$	$j_{burst}$
	external loadings incl. external pressure					internal pressure			
Metallic structures	1.1	1.25	1.25	1.5		1.0	1.0	1.25	1.5
FRP structures (uniform material)	?	1.25	-	1.5		1.0	1.0	1.?	1.5
FRP structures (discontinuities)	-	1.25	-	1.5	1.2				
Sandwich struct.: - Face wrinkling - Intracell buckl. - Honeycomb shear	-	1.25	-	1.5					
Glass/Ceramic structures	-	2.5	-	5.0					
Buckling	-	1.5	-	?					

Experience won, shows up higher risk than usual

Term  $j_{p0.2}$  does not so much fit to actual (relatively brittle) composites!

thermal loading

(ECSS-E-30-10, spacecraft)

## 4 Input of Appropriate Properties for Linear and Non-linear Analysis

### 4.1 Self-explaining Notations for Strength Properties (homogenised material)

		Fracture Strength Properties									
		tension			compression			shear			
		1	2	3	1	2	3	12	23	13	
loading											
direction or plane											
due to material symmetry	9	general orthotropic	$R_1^t$	$R_2^t$	$R_3^t$	$R_1^c$	$R_2^c$	$R_3^c$	$R_{12}$	$R_{23}$	$R_{13}$
	5	UD, ≅ non-crimp fabrics	$R_{  }^t$ NF	$R_{\perp}^t$ NF	$R_{\perp}^t$ NF	$R_{  }^c$ SF	$R_{\perp}^c$ SF	$R_{\perp}^c$ SF	$R_{  \perp}$ SF	$R_{\perp\perp}$ NF	$R_{ \perp}$ SF
	6	fabrics	$R_W^t$	$R_F^t$	$R_3^t$	$R_W^c$	$R_F^c$	$R_3^c$	$R_{WF}$	$R_{F3}$	$R_{W3}$
	9	fabrics general	$R_W^t$	$R_F^t$	$R_3^t$	$R_W^c$	$R_F^c$	$R_3^c$	$R_{WF}$	$R_{F3}$	$R_{W3}$
	5	mat	$R_{1M}^t$	$R_{1M}^t$	$R_{3M}^t$	$R_M^c$	$R_{1M}^c$	$R_{3M}^c$	$R_M^r$	$R_M^r$	$R_M^r$
	2	isotropic	$R_m$ SF	$R_m$ SF	$R_m$ SF	deformation-limited			$R_M^r$	$R_M^r$	$R_M^r$
		$R_m$ NF	$R_m$ NF	$R_m$ NF	$R_m^c$ SF	$R_m^c$ SF	$R_m^c$ SF	$R_m^r$ NF	$R_m^r$ NF	$R_m^r$ NF	

NOTE: \*As a consequence to isotropic materials (European standardisation) the letter R has to be used for strength. US notations for UD material with letters X (direction 1) and Y (direction 2) confuse with the structure axes' descriptions X and Y. \*Effect of curing-based residual stresses and environment dependent on hygro-thermal stresses. \*Effect of the difference of stress-strain curves of e.g. the usually isolated UD test specimen and the embedded (redundancy) UD laminae.  $R_m$  := 'resistance maximale' (French) = tensile fracture strength (superscript t here usually skipped), R:= basic strength. Composites are most often brittle and dense, not porous! SF = shear fracture

## 4 Input of Appropriate Properties for Linear and Non-linear Analysis

### 4.2 Utilization of which Statistical Properties ?

#### 1 Input: DESIGN Stress & Strain Analysis .....Dimensionierung, Struktur-Analyse

*Mean elasticity properties and geometry (thickness, length)*

*to represent mean structural behaviour.*

*Is a necessity in case of (usual) redundant behaviour of the structure*

#### 2 Input: Strength Demonstration (verification)..... ..Nachweis

*One-sided (static and fatigue strength), and two-sided tolerance bands*

*(thickness, E-modulus) have to be considered ...*

#### 3 Input: Stiffness Demonstration

*Due to stiffness requirements → upper and/or lower tolerance limits*

#### 4 A-and B-value Design Allowables (Aerospace) (statistics-based, Mil Hdbk)

*A-values: Application of the military Safe Life Concept*

*B-values: Application of Damage Tolerance Concept (multiple load paths, redundancy).*

NOTE: To achieve a reliable design the so-called *Design Allowable* has to be applied.

It is a value, beyond which at least 99% ("A" value) or 90% ("B" value) of the population of values is expected to fall, with a 95% confidence (on test data achievement) level, see MIL-Hdbk 17.

## 5 Design Limit Loads and Dimensioning Load Cases

### 5.1 Load Cases for the example High Pressure Vessel

---

ESA/ESTEC requirement standard ECSS-30-2 on pressurised hardware says:

As a minimum, any item of a pressurised hardware shall possess, throughout the respective service life of the hardware in the expected operating environments, a strength such to withstand the:

1. *PP (proof pressure) without detrimental deformation;*
2. *DBP (design burst pressure) without experiencing rupture or fibre failure;*
3. *DYL and simultaneous internal pressure multiplied by  $j_{yield}$  without detrimental deformation;*
4. *MDP multiplied by  $j_{yield}$  and simultaneous loads multiplied by  $j_{yield}$  without detrimental deformation;*
5. *DUL and simultaneous internal pressure multiplied by  $j_{ult}$  without experiencing rupture or fibre failure;*
6. *DLP ( $\equiv$  MDP) multiplied by  $j_{ult}$  and simultaneous loads multiplied by  $j_{ult}$  without experiencing rupture or fibre failure;*
7. *DUL and simultaneous external pressure multiplied by  $j_{ult}$  without experiencing rupture or fibre failure when pressurised to the minimum anticipated operating pressure.*

NOTE:  $j_{yield}$  (respectively  $j_{ult}$ ) to be applied on pressure loads and  $j_{yield}$  (respectively  $j_{ult}$ ) to be applied on external or thermal loads can be different. Fos for pressure vessels are higher than for mechanical loading. Positive margins of safety shall be demonstrated by analysis or test or both.

This indicates how one can come from single DLL loadings to high number of load cases (may be hundreds or more). From them, finally, the designer has to sort out a limited number of Dimensioning Load Cases (DLC).

## 5 Design Limit Loads and Dimensioning Load Cases

### 5.2 Dimensioning Load Cases for the example High Pressure Vessel

---

In the case of a metallic High Pressure Vessel from the previously given Load Cases are derived the following **Dimensioning Load Cases (DimLC)**:

- for ductile behaviour the : Yielding-related Load Cases
- for brittle behaviour the : Ultimate-related Load Cases.

**DimLCs are requested in order to:**

- support fast engineering decisions in cases of 'input' changes
- avoid analysis and analysis data evaluation overkill.

**NOTE:** Thinking about the DimLCs improves

- understanding of structural behaviour and
- engineering judgement.

## 6 Link *Deterministic to Probabilistic Design*

### 6.1 Procedures

---

The usual way to „verify a design“ is to show by computation,  $MS > 0$ ,

the resistance of a structure is higher than the loading  
(for critical cross section loads or stress combinations or ...).

→

$\mathfrak{R}$

$\mathfrak{R}$

In the reliability analysis, this way is more complex.

It's objective is

the evaluation of a probability of occurrence of a given failure state

$p_f < \text{admissible } p_f$ ,

or of a survival probability = reliability  $\mathfrak{R} > \text{required}$  .

$$\mathfrak{R} = 1 - p_f$$

## 6 Link *Deterministic to Probabilistic Design*

### 6.2 Analysis Goal and Optimisation

---

Goals in *Structural Analysis*

in order to achieve *Structural Integrity* :

- **Mass minimisation of the structure** (mathematical formulation)  
in the prescribed design space  
wrt *side constraints* such as *cost, project deadlines, manufacturing*  
and *NDI needs, risk* (amount of failure cost • probability this failure occurs)
- **Prediction of structural behaviour and strength analysis** (*Design Verification*)  
→ result : a set of nominal (mean) design parameters.

## 6 Link Deterministic to Probabilistic Design

### 6.3 Analysis Goal and Optimisation

#### Goals of Structural Analysis

- **Mass minimisation of the structure** (mathematical formulation) in the prescribed design space  
wrt *side constraints* such as *cost, project deadlines, manufacturing and NDI needs, risk* (amount of *failure cost* • *probability* this failure occurs)
- **Prediction of structural behaviour and strength analysis (Design Verification)**  
→ result : a set of nominal (mean) design parameters.

#### Deterministic optimisation of a structural model

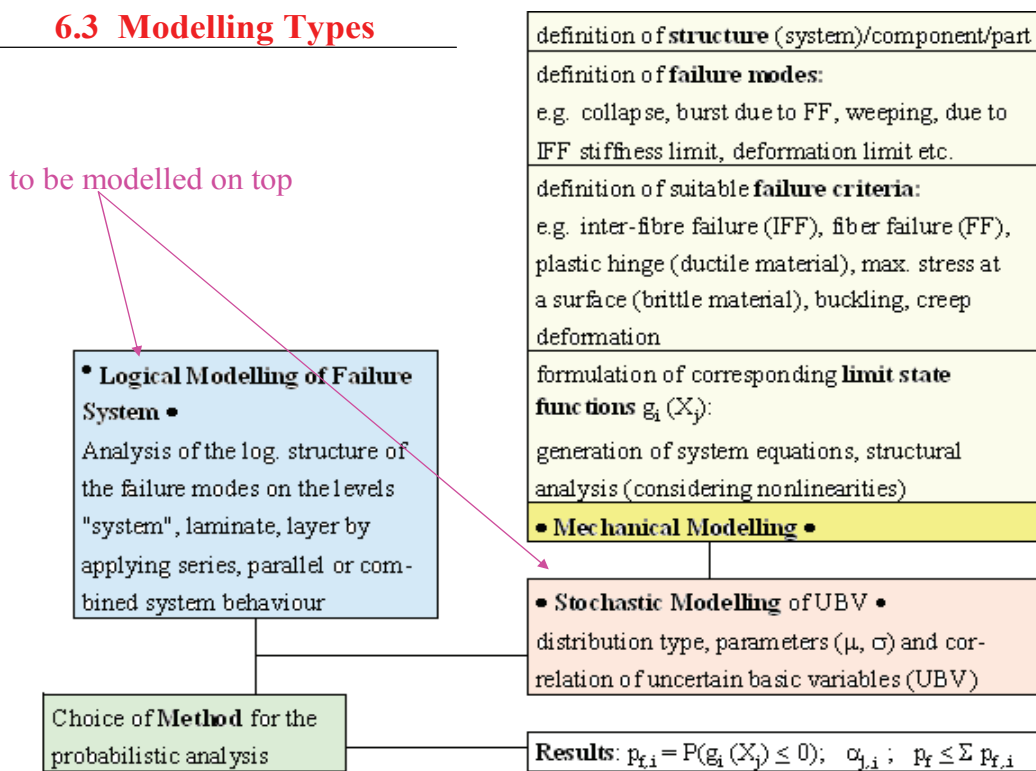
→ set of mean design parameters  
with which the (failure) limit states are not reached by a not quantifiable distance in the frame of the scatter covering deterministic FoS

#### Stochastic (probabilistic) optimisation (dream becomes slowly true)

→ set of mean design parameters ('most probable failure point')  
however with  $p_f$  as a measure for this distance  
and directly considering the scatter of all design parameters.

## 6 Link Deterministic to Probabilistic Design

### 6.3 Modelling Types



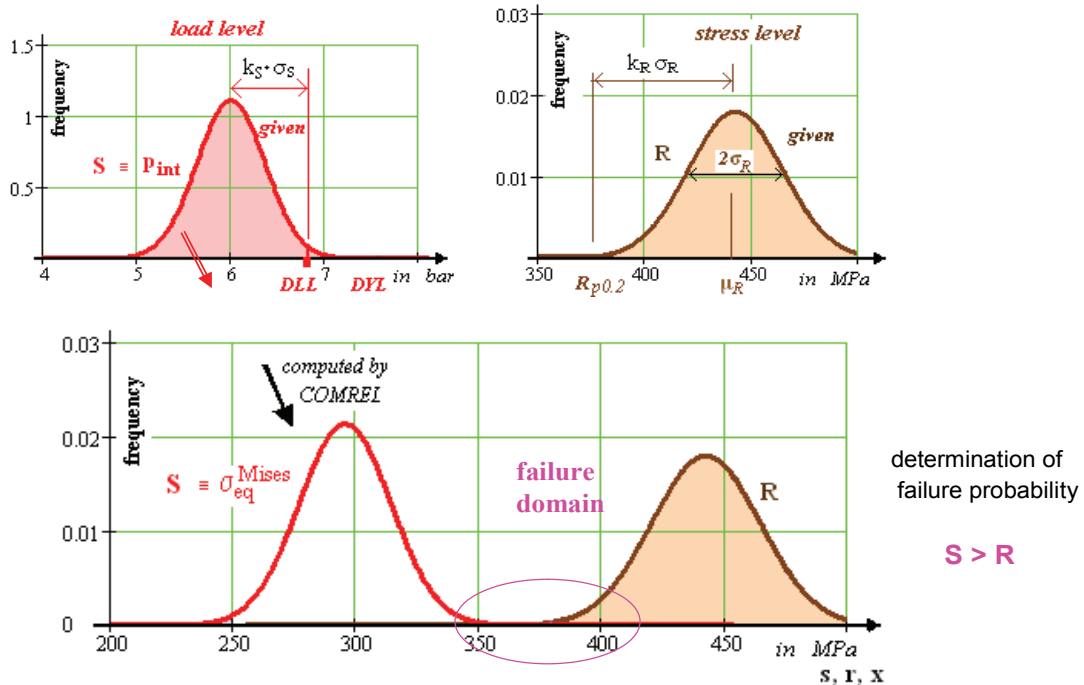
UBV = uncertain basic variable

**Utilization**  
for lay-out and design verification



## 6.4 Example: Pressure Vessel (e.g. ARIANE 5 Booster)

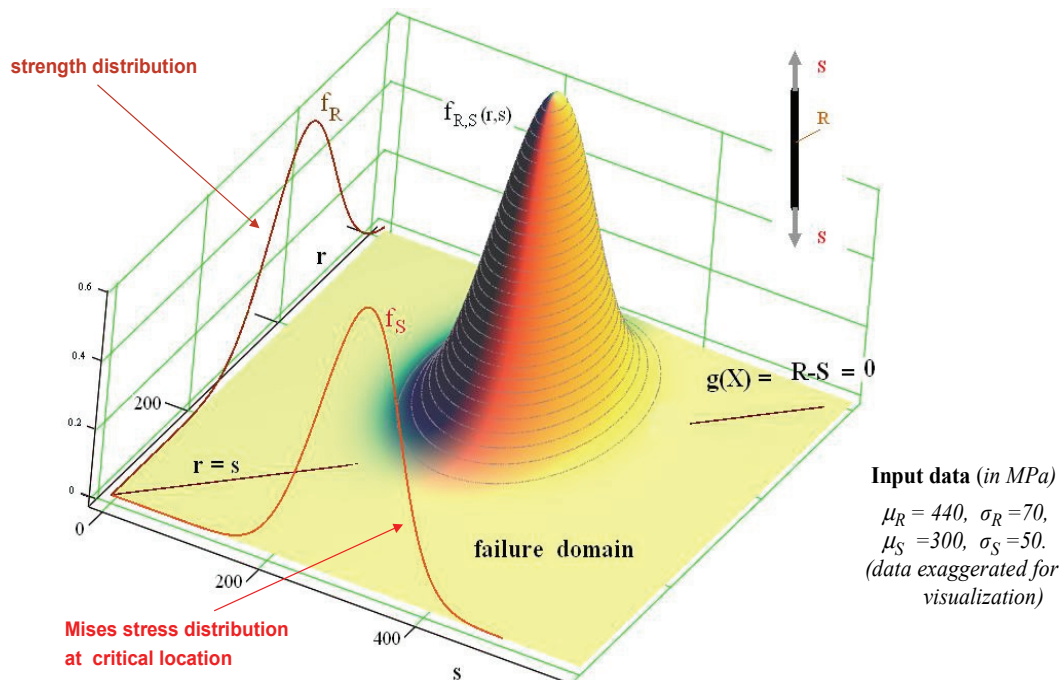
### 1 Input



The larger the distance between the stress and the strength distribution the larger become MS and the survival probability = reliability  $\mathcal{R} = 1 - p_f$ .

## 6.4 Example : Tension Rod

### 2 Visualization of the Determination of the Failure Probability



**NOTE:** Failure probability corresponds to percentage of the volume of the probability hill vertically cut off by  $r = s$  or  $g(X) = 0$ .

## Example: Pressure Vessel

### 3 Effect of different Strength Distribution on Failure Probability

$j$	$X_j$	$\mu_j$	$\sigma_j$	$f_j$	$CoV_j$
1	$\sigma_{eq}^{Mixed}$	295 MPa	18.6 MPa	ND	6.3 %
2	$R_{p0.2}$	442 MPa	22.1 MPa	ND	5 %

input

$\alpha_j$	$x_j$ of $x^*$	$p_f$
-0.66	356 MPa	$1.8 \cdot 10^{-7}$
+0.76	356 MPa	$10^{-8}$

results

MS = -3.7%

$j$	$X_j$	$\mu_j$	$\sigma_j$	$f_j$	$CoV_j$
1	$\sigma_{eq}^{Mixed}$	295 MPa	18.6 MPa	ND	6.3 %
2	$R_{p0.2}$	440 MPa	19.5 MPa	ND	4.4 %

$\alpha_j$	$x_j^*$	$p_f$
-0.69	363 MPa	$3.2 \cdot 10^{-8}$
0.73	363 MPa	$10^{-8}$

MS = -1.1%

Design Verification Onset of Yielding:  $p_f = 1.8 \cdot 10^{-7} \leq 2 \cdot 10^{-7} = \text{admissible } p_f$

O.K.

**NOTE: Despite lowering the mean strength value the failure probability  $p_f$  is reduced due to the reduction of the Coefficient of Variation !**

## Further application: Ariane 5 Launcher, Booster

### 4 Influence of Reduction of Manufacturing Tolerance

Thickness tolerance

Former :  $t = 8.2 \pm 0.20 \text{ mm}$

Improved manufacturing :  $\pm 0.05 \text{ mm}$ .

Reduction in scatter permits

– keeping the same theoretical reliability value  $\mathfrak{R} = 1 - p_f = 1 - 5 \cdot 10^{-6}$  -

New nominal thickness :  $t = 8.1 \pm 0.05 \text{ mm}$

► mass reduction of? kg  
at the same reliability level!

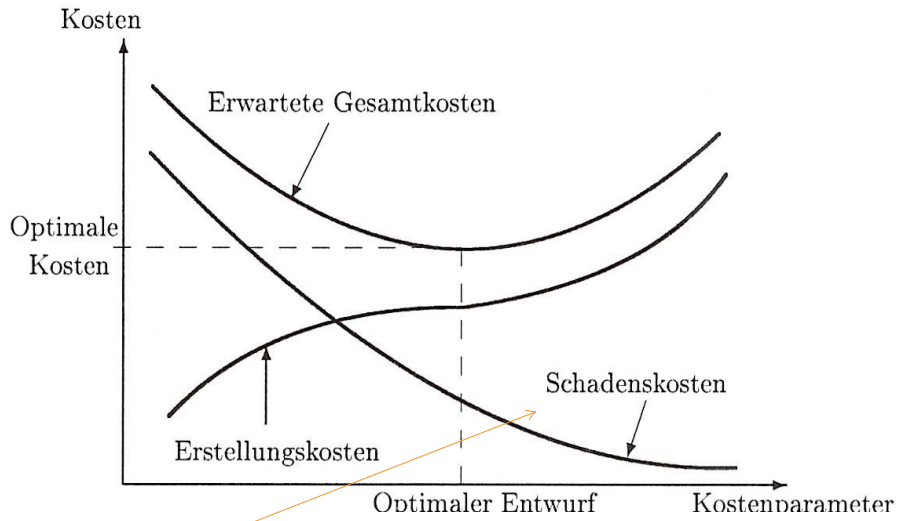
+ Fuel savings ???

## Summary of experience with Deterministic + Probabilistic analyses

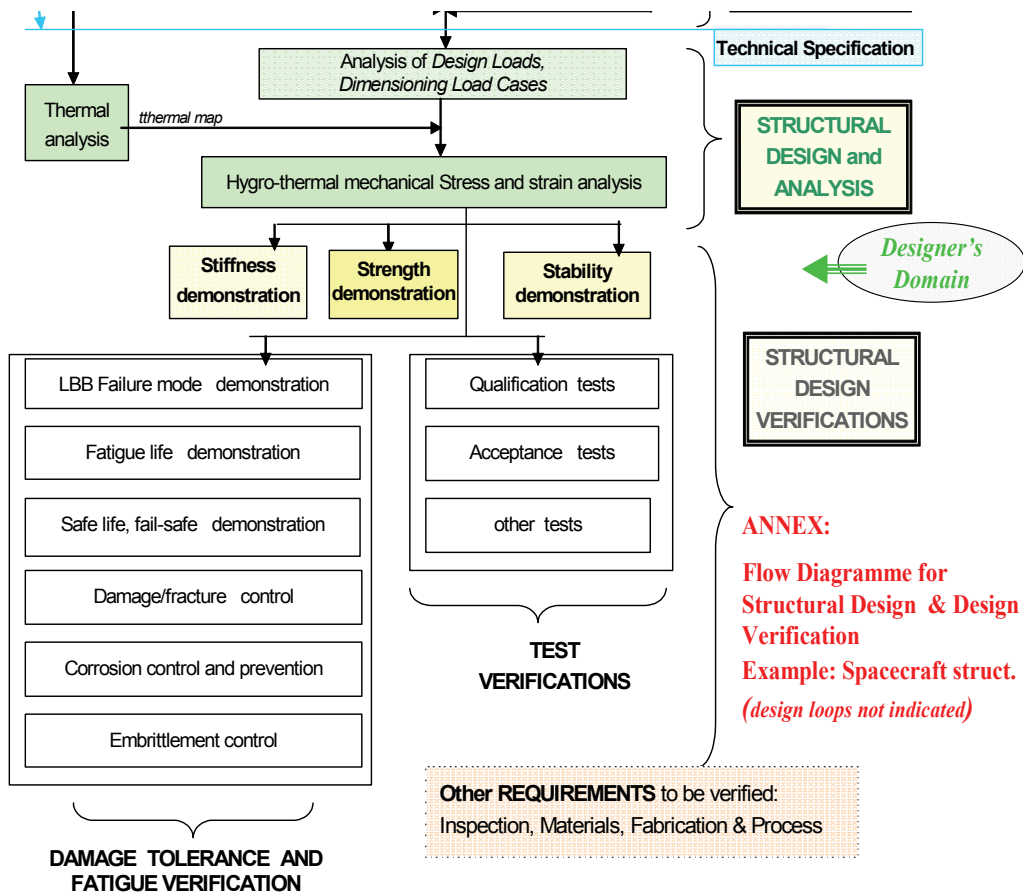
1. A *robust reliable design* or the robustness to later changes of the design parameters with identification of the most sensitive design parameters is a need in order to save life cycle costs
2. Deterministic optimization may deliver fully non-robust, even non-stable designs and probably not the real optimum. A counteraction to this will stand for more effort for the designer: Execution of several common deterministic optimization runs regarding the scatter of the design parameters
3. Stochastic optimization means a little more input effort for the designer and more effort for the computer than in the deterministic case. However, just a stochastic robust design informs about the risk
4. Failure probability  $p_f$  does not dramatically increase if a Margin of Safety turns slightly negative. A local safety measure of e.g.  $MS = -1\%$  should be no problem in *design development*.  
The MS value does not outline the risk or the failure probability. Therefore, do not overreact by re-designing but apply a ‘Think (about) Uncertainties’ attitude by recognizing the main driving design parameters and by reducing the scatter (uncertainty) of them. This highly pays off!
5. Essential question wrt all uncertainties is whether these increase the risk to an unacceptable level or not. The system authority provides with a value for a failure probability  $admissible p_f \geq p_f$  or a reliability  $\mathfrak{R} \geq required \mathfrak{R}$ .
6. Physics have to be modelled accurately in the analysis part *Mechanical Modelling*. All Dimensioning Load Cases have to be accounted for. The choice of the task-corresponding stress-strain curve has to be carefully performed (min or mean or max). The choice of an engineering  $(\sigma, \epsilon)$  or a true  $(\sigma, \epsilon)$  curve depends on the output of the utilized code
7. Both, an increasing mean value and a decreasing standard deviation lower  $p_f$
8. Final comment:  
*Theory* ‘only’ creates a model of the reality, and *Experiment* is ‘just’ one realization of the reality. Experimental results can be far away from the reality like an inaccurate theoretical model.  
So, find a compromise to cost-optimally achieve an improved *Analysis-Test Verification* procedure for the robust design at hand.

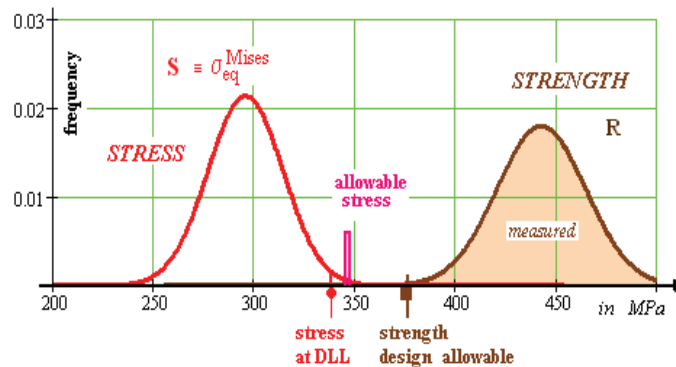
ANNEXES:

Cost → Minimum  
Quality → Maximum



Risk = amount of cost incurred in the case of later failure x probability of its occurrence





*Actual safety concept in aerospace: A deterministic concept that enlarges the deterministic load (stresses) and causes thereby a distance to the load resistance (strengths). This difference represents the required positive margin of safety. The other deterministic concept, the safety concept of Allowable Load (Stresses), is not applied in aerospace anymore. This concept reduces the load resistances (strength). Just the load enlarging concept delivers accurate margins in case of non-linear analysis*

**Fig:** Essential notions in deterministic design

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